Characteristics of Unsteady Flows around a Circular Cylinder
with Simultaneous Multipoint Measurements of Pressure

by

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Abstract
This study aims to investigate the unsteady characteristics of flows close to the critical state around a circular cylinder experimentally by simultaneous multipoint pressure measurements. For the measurements of pressure distribution, 128 holes are made spirally around the cylinder and each hole is connected through a vinyl tube to a semiconductor pressure sensor. For the range of Reynolds number, 1.4×10⁵ to 5.5×10⁵, the following quantities are computed and analysed: unsteady and time-averaged distributions of pressure coefficients, instantaneous and time-averaged drag and lift coefficients, perturbation of the lift coefficient, and Strouhal number, etc. As results, characteristics of the unsteady flows have been captured in subcritical, critical, and supercritical states. Above all, the characteristics in the critical range have been presented.

Keywords: Circular cylinder, Multipoint pressure measurements, Critical state

1. Introduction

The cylinder is used in various fields including the engineering field because it is a basic shape in all structural things, and also flows about the cylinder have been investigated from the view point of fluid mechanics1-4), for example, on detailed flow structure with separation and consequent flow-induced vibration.

At the present day, owing to the progress of large-scale computers, by use of the CFD (Computational Fluid Dynamics)5-6) it became possible to capture the structure of three-dimensional unsteady flows in detail and also to apply the computational results to engineering designs, but the computational results depend on the accuracy of numerical methods, grid fineness, and boundary conditions, etc. Therefore for flows including severe conditions such as transition, turbulence, and separation, the validation study would still need the proper experimental data.

Regarding critical flows with the severe conditions about the simple shape of a circular cylinder, study by Roshko7) is classical and his experimental data for surface pressure distribution have been cited in many text books, but in mean values. Unsteady property of the flows has not been made fully clear still now.

In the series of authors’ researches8,9), unsteady flows have been investigated in the vicinity of the critical state around a circular cylinder experimentally by use of simultaneous multipoint pressure measurements10). In these experiments, pressures at 128 holes are simultaneously measured, and the case with such multipoints is very rare. In the first trial8), reliability was maintained only for Reynolds number less than 5×10⁵ because of ability limit of pressure sensors used. In the next step9), the range of Reynolds number was expanded up to 5.5×10⁵ by raising the performance with new sensors. Thus unsteady flows have been analyzed to grasp the characteristics in subcritical, critical, and supercritical states.

In this paper, in addition to the previous results, enhanced data analysis such as FFT and perturbation is shown to present the characteristic behavior in the vicinity of the critical state more in detail by simultaneous multipoint pressure measurements.

2. Experimental Setup

2.1 Large-scale low-speed wind tunnel

In the experiments the large-scale low-speed wind tunnel of Tokai University has been used. The wind tunnel is
single-return closed-circuit type with the open jet test section (Fig.1), where the ground board is set up between the nozzle exit and the collector of the wind tunnel (Fig.2). The dimension of the test section is 1 m high, 1.5 m wide and 2 m long, and the wind speed range is 0 to about 40 m/s.

2.2 Device for simultaneous multipoint pressure measurements

Here the semiconductor pressure transducer, sensor MS4515 (Measurement Specialties Inc.), is adopted, and its output range in Voltage is shown in Fig.3. There are 2 ports on the pressure sensor: one is connected through the vinyl chloride tube with length of 1.5 m to a hole for pressure measurement, and the other is open to the atmospheric pressure. The difference pressure between the two ports is obtained as the output voltage when 5 V-DC is supplied. Figure 4 shows the device for multipoint pressure measurements. This measurement device consists of two cases, and each case stores 4 boards, on each of which 16 pressure sensors are arranged, so that pressure at 128 points can be measured at once.

The data from the transducers are acquired through the terminal board by a PC with two 64-channel A/D converters, so that distributions of surface-pressure data on a cylinder can be measured at the same time. Figure 5 shows the experimental setup of multipoint pressure measurements.

Table 1 Speed and Re number

<table>
<thead>
<tr>
<th>Speed (m/s)</th>
<th>Re (×10^5)</th>
</tr>
</thead>
<tbody>
<tr>
<td>10</td>
<td>2.86~3.57</td>
</tr>
<tr>
<td>15</td>
<td>2.86~3.57</td>
</tr>
<tr>
<td>20</td>
<td>2.86~3.57</td>
</tr>
<tr>
<td>25</td>
<td>2.86~3.57</td>
</tr>
<tr>
<td>30</td>
<td>2.86~3.57</td>
</tr>
<tr>
<td>35</td>
<td>2.86~3.57</td>
</tr>
</tbody>
</table>

First, from Fig. 12, since CDp abruptly decreases at about $\theta = 84^\circ$, the flow field is in the subcritical state for Re $(2.86~3.57) \times 10^5$, it is comprehended that the flow field is therefore aspect ratio $l/d$ is 4.63. 128 holes for pressure measurements were made spirally around the cylinder surface in order to avoid the flow interference between adjacent holes. There are 128 points on the cylinder model are attached to the parallel wall surface. In order to measure the pressure at 128 points can be measured at once. The dimension of the test section is 1 m high, 1.5 m wide and 2 m long, and the wind speed range is 0 to about 40 m/s.
2.3 Model of circular cylinder

First, the diameter of the cylinder model was decided so that critical Reynolds number about $3 \times 10^5$ should be obtained at wind speed 20 m/s, and then the polyvinyl chloride pipe with a standard size close to the above diameter was adopted. The dimension is 0.216 m in diameter (d) and 1.0 m in length (l), therefore aspect ratio $l/d$ is $4.63$. 128 holes for pressure measurements were made spirally around the cylinder surface to avoid the flow interference between adjacent holes.

Figure 6 shows setup of the circular cylinder model within the test section of the wind tunnel, where both ends of the cylinder model are attached to the parallel wall surface. In order to avoid suffering from effects of the boundary layer on the test-section wall, the pressure holes are created except 0.1 m portion for each end.

2.4 Experimental conditions

Wind speed ($U$) is set at interval of 5 m/s from 10 m/s to 35 m/s, and at maximum speed 38 m/s, totally in 7 patterns (see Table 1), and the resultant range of Reynolds number (Re) $1.4 \times 10^5 < Re < 5.5 \times 10^5$ includes the critical state $Re = 3 \times 10^5$. In pressure measurements, the sampling frequency is 400 Hz with number of sampling points 17000, so that the total sampling range is 42.5 sec.

<table>
<thead>
<tr>
<th>Wind Speed U (m/s)</th>
<th>Reynolds Number Re $\times 10^5$</th>
</tr>
</thead>
<tbody>
<tr>
<td>10</td>
<td>1.43</td>
</tr>
<tr>
<td>15</td>
<td>2.14</td>
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<tr>
<td>20</td>
<td>2.86</td>
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<tr>
<td>25</td>
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<td>35</td>
<td>5.00</td>
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<tr>
<td>38</td>
<td>5.43</td>
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3. Results and Discussion

Figure 7 shows the distributions of time-averaged surface pressure coefficients ($C_p$) during last 5 seconds for each Reynolds number, and Fig. 8 does those of RMS (Root Mean Square) for time history of $C_p$ at each location and condition in Fig.7. In both figures the horizontal axes are for angle $\theta$ from the stagnation point. Comparison with typical $C_p$ distributions is displayed in Fig.9. Figure 10 shows the time history of pressure-drag coefficients $C_{dp}$ and pressure-lift coefficients $C_{lp}$ computed from instantaneous $C_p$ distributions for each flow condition of $U=10, 15, 20, 25, 30, 35$ m/s. Figure 11 shows typical results of FFT analyses for time histories of $C_{dp}$ and $C_{lp}$ in Fig.10. Figure 12 shows change of $C_{dp}$ calculated from the time-averaged $C_p$ distributions during last 5 seconds, with regard to the Reynolds number, where error bars are from maximum and minimum values of instantaneous $C_{dp}$ during total 42.5 seconds. Figures 13 and 14 show, respectively, changes of RMS of $C_{dp}$ (perturbation of pressure-drag coefficients) and RMS of $C_{lp}$ (perturbation of pressure-lift coefficients), calculated from time history of instantaneous $C_{dp}$ and $C_{lp}$ (Fig.10) during last 5 seconds, with regard to the Reynolds number.

Figures 15 and 16 show the relations between estimated Strouhal number (St) and Reynolds number. Here Strouhal number $St = f d/U$ is estimated by use of peak frequency (frequency with peak amplitude) $f$ in the FFT analysis. Strouhal numbers in Fig.15 are calculated from the FFT for the time history of pressure in voltage at local points (a)$\theta=84^\circ$, (b)$\theta=118^\circ$ and (c)$\theta=146^\circ$, while Strouhal numbers in Fig.16 are computed from the FFT for the time history of $C_{lp}$.

3.1 Time-averaged $C_p$ distributions

First, from Fig.12, since $C_{dp}$ abruptly decreases at about $Re=(2.86-3.57)\times10^5$, it is comprehended that the flow field is in the critical state for $Re=2.86\times10^5, 3.57\times10^5$, and therefore the flow field is in the subcritical state for $Re \leq 2.14\times10^5$ and in the supercritical state for $Re \geq 4.29\times10^5$. 
In Fig.7, for the subcritical states for Re≤2.14×10^5, Cp decreases from θ=0° to the vicinity of θ=70°, then recovers up to 80°, and afterwards takes almost constant value Cp ≈ 1.0. Correspondingly RMS distributions (Fig.8) have the maximum values at about θ=75°. It is regarded that the laminar boundary layer is separated at about θ=80° in the upstream side of the cylinder top.

However, for the critical states for Re=2.86×10^5 and 3.57×10^5, Cp distributions become unsymmetrical between θ>0° and θ<0°. From the observation in the time history of CDp and CLp (Fig.10) that CLp is not zero in average for the corresponding Reynolds number, it is presumed that alternate release of vortices are not made equally for θ>0° and θ<0°, which is surely observed in the animation for time-dependent Cp distributions.

In the supercritical range where the Reynolds number is made higher to Re≥4.29×10^5, Cp distributions become again almost symmetrical for θ>0° and θ<0°. Cp decreases abruptly down until about θ=80°, it is gradually recovered during 80°≤θ≤105°, afterwards Cp increases at a burst for 105°≤θ≤130°, and it is almost constant for θ>130°.

In the critical and supercritical states the RMS distributions have two peaks for θ>0°, low peaks at about θ=90° and sharp peaks at about θ=105°~115°, in the vicinity of the inflection points of Cp curves, and relatively middle values during about 20° after the sharp peaks. Along with established explanation, this Cp pattern is due to the phenomenon that the boundary layer once separates at about θ=90° (low RMS peak), the layer becomes turbulent, reattaches at about θ=105°~115° (sharp RMS peak), flows along the cylinder surface, and the turbulent boundary layer is separated with vortex release in the downstream side of the cylinder top. The RMS distributions for θ>0° indicate that the reattachment points are located more downstream for the critical state than for the supercritical state.
values, but the tendency is very similar.

3.2 Time history of $C_{Dp}$ and $C_{Lp}$

About the pressure-drag coefficients in the time history shown in Fig.10, the instantaneous $C_{Dp}$ values almost agrees, in time average, with the mean $C_{Dp}$ values in Fig.12 computed from the time-averaged $C_p$ distribution.

About the pressure-lift coefficients, the time history shows some characteristics. The wave shape can be regarded that oscillations with high frequency are added on a global undulation with low frequency. In the case of $Re=1.43 \times 10^5$ the amplitude of high-frequency oscillation is largest and effect of global undulation is limited. The reason would be that alternate appearance of low pressure portions by vortex shedding generates the large amplitude of lift oscillations since the laminar boundary layer is separated in the upstream side of the cylinder top. As the Reynolds number is made higher up to $2.86 \times 10^5$, amplitude of the high-frequency component becomes smaller, and the low-frequency undulation becomes more visible, and is visibly remarkable for $Re \leq 3.57 \times 10^5$.

Actually in FFT analyses shown in Fig.11, for $Re=2.14 \times 10^5, 2.86 \times 10^5$ there appear two peak frequencies: a low peak frequency and a normal peak frequency corresponding to the frequency of vortex shedding. The amplitude of the low frequency becomes larger as the Reynolds number increases up to $Re=2.86 \times 10^5$. For $Re=3.57 \times 10^5$ there are many lower and higher peak frequencies than the normal peak frequency. Here the normal peak frequency is specified by Strouhal number about 0.2. In Figs.11 and 16 it is observed that various frequencies for oscillations of $C_{Lp}$ appear for $Re \geq 3.57 \times 10^5$ where separation of the turbulent boundary layer occurs.

It is remarkable in Fig.10 that at $Re=2.86 \times 10^5$ and $3.57 \times 10^5$ in the critical state, $C_{Lp}$ is not zero in average, which is considered that the wind leans to some direction.

Fig.9 Comparison with typical $C_p$ distributions

Fig.10 Time history of instantaneous $C_{Dp}$ and $C_{Lp}$
As mentioned before, since CDp abruptly decreases at about 3.3
separated in the upstream side of the cylinder top and the vortex
subcritical range, because the laminar boundary layer is
and l/d=\infty, respectively, and at about Re=3\times10^5 \sim 4\times10^5 the
flow transition s3). In the present case of l/d=4.63 with both
Re=(2.86\sim3.57)\times10^5, the flow field is in the critical state,
therefore the flow field is in the subcritical state for Re
Re=3\times10^5, it is asserted that at \U=20, 25 m/s, i.e., Re=(2.86\sim3.57)\times10^5, the flow field is in the critical state, and therefore the flow field is in the subcritical state for Re\leq2.14\times10^5 and in the supercritical state for Re\geq4.29\times10^5.

Regarding the perturbation of drag coefficients shown in
Fig.13, RMS of CDp is very high for Re=1.43\times10^5 in the
subcritical range, because the laminar boundary layer is
separated in the upstream side of the cylinder top and the vortex
shedding does cause large drag oscillations. There is tendency
that the drag perturbation is gradually made lower with increase
of Reynolds number, except for a high value at Re=2.86\times10^5 in
the critical range. Regarding the perturbation of lift coefficients
shown in Fig.14, RMS of CLp is very low for Re\geq4.29\times10^5 in
the supercritical range, because the turbulent boundary layer is
separated in the downstream side of the cylinder top and the vortex
shedding does not cause large lift oscillations. It is
notable that the change of lift perturbation indicates the same
tendency as the change of the drag coefficient. Thus it is
confirmed that for both drag and lift coefficients, the perturbation is locally increased in the region from the
subcritical to the critical states.

Fig.11 Results of FFT analyses for CDp and CLp

3.3 Change of CDp, RMS of CLp and St with regard to Re

About drag coefficients of the circular cylinder, for Re \leq 2\times10^5, CDp=0.74 and CDp=1.2 in the case of aspect ratio l/d=5
and l/d=\infty, respectively, and at about Re=3\times10^5 \sim 4\times10^5 the
flow transitions\(^\text{3}\). In the present case of l/d=4.63 with both
cylinder ends attached to the parallel walls, where wall
interference occurs, CDp=0.8\sim1.1 for Re\leq2.86\times10^5 in Fig.12.
As mentioned before, since CDp abruptly decreases at about
Re=3\times10^5, it is asserted that at \U=20, 25 m/s, i.e., Re=(2.86\sim3.57)\times10^5, the flow field is in the critical state, and therefore the flow field is in the subcritical state for Re\leq2.14\times10^5 and in the supercritical state for Re\geq4.29\times10^5.

Regarding the perturbation of drag coefficients shown in
Fig.13, RMS of CDp is very high for Re=1.43\times10^5 in the
subcritical range, because the laminar boundary layer is
separated in the upstream side of the cylinder top and the vortex

Fig.12 Change of CDp with regard to Re

Fig.13 Perturbation of CDp with regard to Re

Fig.14 Perturbation of CLp with regard to Re
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About change of Strouhal number with regard to Reynolds number, Generally it is stated\(^1\) that in range of \(250 < \text{Re} < 3 \times 10^5\), the Strouhal number for the vortex release is about \(S_t = 0.19 \sim 0.21\), and it rises up to \(S_t = 0.4 \sim 0.5\) in the Re range from the critical to the supercritical states.

Figure 15 shows the change of Strouhal number estimated by dominant frequencies (frequency with dominant peak amplitude) in the FFT for pressure history in output voltage during total 42.5 seconds and last 5 seconds at local points (a)\(\theta = 84.375\)°, (b)\(\theta = 118.125\)°, and (c)\(\theta = 146.25\)°. In three cases the Strouhal number is about 0.2 for \(\text{Re} \simeq 2.86 \times 10^5\) in the subcritical range, which corresponds to general statement\(^2\) previously mentioned. At (a)\(\theta = 84.375\)° in the vicinity of the separation point of the laminar boundary layer in the upstream of the cylinder top, the pressure frequency abruptly increases for \(\text{Re} \simeq 5 \times 10^5\) in the supercritical region. On the other hand, at (b) \(\theta = 118.125\)° in the vicinity of the separation point of the turbulent boundary layer, the pressure frequency increases at \(\text{Re} = 2.86 \times 10^5\), from the subcritical to the critical region, but \(S_t\) depends on the duration of the FFT, which indicates that time dependency of the phenomenon is strong. At (c) \(\theta = 146.25\)° in the separation region, the pressure frequency is always large at \(\text{Re} = 2.86 \times 10^5\), which implies that wake region suffers from influence of pressure perturbation at the turbulent separation point in the region from the subcritical to the critical states.

In Fig.16 change of Strouhal number for Reynolds number is shown where \(S_t\) is estimated by the peak frequency in the FFT results (Fig.11) for the time history of \(C_{lp}\) (Fig.10), because the frequency of lift force directly corresponds to that of the vortex release. As is observed in Fig.11, number of peak frequencies increases for \(\text{Re} \simeq 3.57 \times 10^5\), from the critical to the supercritical region. Therefore in the graph Strouhal number based on the dominant frequency is marked by circle, with peak frequencies shown by triangles. It is confirmed that the change of Strouhal number shows general tendency\(^2\) previously stated.

4. Conclusions

Characteristics of unsteady flows around the circular cylinder have been captured experimentally by simultaneous multipoint pressure measurement. As results the following conclusions have been obtained:

1) Characteristics of unsteady flows have been shown in subcritical, critical, and supercritical states. Above all, the characteristics in the critical range have been presented.

2) The simultaneous multipoint pressure measurements have been almost successful.

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**Fig.15 Change of St based on FFT for pressure in voltage**

\(\theta = 84.375\)° (at laminar separation point)

\(\theta = 118.125\)° (at turbulent separation point)

\(\theta = 146.25\)° (in separation region)

**Fig.16 Change of St based on FFT for \(C_{lp}\)**
REFERENCES


